## U-3ARC <br> TRAINING WEBINAR \#22

## HEAT BALANCE OF A COLD ROOM

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## Summary

## General

## Introduction

Determination of the time base
Calculation of the various balance sheet items
Sum of contributions from all items
Calculation of cooling capacity

## General

## Food spoilage process

In some regions of the world, half of the food available is lost between the period of production and that of consumption. The most important destructive agents:
Rodents, insects, microorganisms (fungi and bacteria)
Humans through improper handling (due to ignorance or negligence)
Place the products in places favoring various alterations

## General

## Food spoilage process

Two main food groups according to their origin
Products of vegetable origin:
$\square$ Live products before and even during the storage period Animal products:
$\square$ Products killed prior to their use
$\checkmark$ Perishables have a diversity of physical properties and chemical compositions
$\checkmark$ All contain water, carbohydrates, proteins, mineral salts and vitamins
$\checkmark$ Water is the major constituent
Main factor of product alterability

## General

## Food spoilage process

Agents responsible for food spoilage

## Enzymes:

Water-soluble protein biocatalysts produced by living cells that control the biochemical reactions of degradation of the quality of food products

## Microorganisms:

Bacteria and/or fungi which are present on the surface or in the cavities (or folds) of the constituent organs of plants and animals Other microorganisms can be brought by the manipulations undergone by the products
Microorganisms can spoil food by growing on them and breaking down their constituent substances

## General

## Incompatibilities due to odors and ethylene

The volatile compounds responsible for the aroma of certain products can be fixed by other neighboring products and give them an undesirable odor or taste.
Example:
Avoid storing:

- Apples with potato
- Citrus with other products.

The smell of apples and citrus is easily fixed by butter, meat and eggs Apples take on an unpleasant color and smell when stored with potato
[ "Living" Plant Organs Are Capable of Ethylene Production at Different Levels
Ethylene is an odorless gas and even in small quantities shortens the life of oranges that synthesize it or those in their vicinity by accelerating their senescence or ripening for fruits

## General

## Incompatibilités dues aux odeurs et à l'éthylène

|  | Ban ane | Beu re | Bœ uf | $\begin{aligned} & \text { Cho } \\ & \text { u } \end{aligned}$ | Fro mag e | Lan gou ste | Lard | $\begin{aligned} & \text { Mo } \\ & \text { uto } \\ & n \end{aligned}$ | ©uf <br> s | $\begin{array}{\|l\|} \hline \text { Ora } \\ \text { nge } \end{array}$ | pêc he | $\begin{array}{\|l} \hline \text { Po } \\ \mathrm{mm} \\ \mathrm{e} \end{array}$ | Po mm e de terr e | $\begin{aligned} & \text { Por } \\ & \text { c } \end{aligned}$ | $\begin{array}{\|l\|} \hline \text { Pru } \\ \text { ne } \end{array}$ | Rais in |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Banane | - | 0 | 0 | N | 0 | 0 | 0 | 0 | 0 | N | N | N | N | 0 | N | 0 |
| Beure | 0 | - | 0 | N | LR | R | LR | 0 | 0 | R | LR | N | N | 0 | 0 | 0 |
| Bœuf | 0 | 0 | - | N | LR | LR | LR | 0 | 0 | N | 0 | R | LR | 0 | 0 | 0 |
| Chou | N | N | N | - | N | N | N | N | N | N | LR | LR | LR | N | LR | LR |
| Fromage | 0 | LR | LR | N | - | N | 0 | LR | N | N | LR | N | LR | LR | LR | LR |
| Langouste | 0 | R | LR | N | N | - | LR | LR | LR | N | LR | N | N | LR | LR | 0 |
| Lard | 0 | LR | LR | N | 0 | LR | - | 0 | 0 | N | 0 | N | 0 | 0 | 0 | 0 |
| Mouton | 0 | 0 | 0 | N | LR | LR | 0 | - | 0 | N | 0 | N | LR | 0 | 0 | 0 |
| Cufs | 0 | 0 | 0 | N | N | LR | 0 | 0 | - | N | LR | N | N | 0 | LR | 0 |
| Orange | N | N | N | N | N | N | N | N | N | - | 0 | 0 | 0 | N | 0 | 0 |
| Pêche | N | LR | 0 | LR | LR | LR | 0 | 0 | LR | 0 | - | 0 | 0 | 0 | 0 | 0 |
| Pomme | N | N | R | LR | N | N | N | N | N | 0 | 0 | - | LR | N | 0 | 0 |
| Pomme de terre | N | N | LR | LR | LR | N | 0 | LR | N | 0 | 0 | LR | - | LR | 0 | 0 |
| Porc | 0 | 0 | 0 | N | LR | LR | 0 | 0 | 0 | N | 0 | N | LR | - | 0 | 0 |
| Prune | N | 0 | 0 | LR | LR | LR | 0 | 0 | LR | 0 | 0 | 0 | 0 | 0 | - | 0 |
| Raisin | 0 | 0 | 0 | LR | LR | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | - |
| 0 | Can be together without risk of mutual contamination |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| LR | Slight risk of mutual contamination |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| R | Risk of mutual contamination |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| N | Cannot be stored together: very likely mutual contamination |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |

## Thermal balance

## Introduction

The sizing of an installation begins with the calculation of the cooling capacity required. This power represents the energy that must be removed from the atmosphere in a given time to maintain the room at the desired temperature. This energy comes from several sources which are:
The walls
$\square$ Products

- Ventilation
- Air renewal
] Machines
Lighting
The occupants
In order to estimate the quantities helt relased by all these sources, we carry out a detailed heat balance item by item, setting a time base.


## Thermal balance

## Determination of the time base

$\square$ Cold rooms for the conservation of fresh or frozen products, the calculations are made for a 24-hour day, i.e. $24 \times 3600=$ 84,400 seconds.
$\square$ Applications whose use is cyclical (for example: cooling tunnel, dryer, cold room for rapid freezing, oven, etc.), the base time of the calculation will correspond to the cycle time. Example :
Rapid cooling tunnel whose daily use is done in 4 cycles of 3 hours. We will carry out the assessment over one cycle (3 hours), i.e. a time base of: $3 \times 3600=10800$ seconds.

## Thermal balance

## Calculation of the various balance sheet items

## Position 1: conduction through the walls

The amount of heat brought by conduction through the walls is given by:

$$
\mathrm{Q} \text { walls }=\mathrm{K} . \mathrm{S} \cdot \Delta \theta \cdot \mathrm{t}
$$

With
Q walls: heat provided by conduction in Joules [J]
K : overall wall exchange coefficient [W.m-2. ${ }^{\circ} \mathrm{C}-1$ ]
S: total wall area [m2]
$\Delta \theta$ : temperature difference between the outside temperature and the inside temperature $\left[{ }^{\circ} \mathrm{C}\right]$
t : basic calculation time [s]

## Thermal balance

## Calculation of the various balance sheet items

## Example :

Consider that our apple preservation chamber has the following dimensions:
Length 20 m
Width 10 m

## Height 6m

$>$ Total surface of the walls of: $2 \mathrm{x}(20 \times 6+10 \times 6)+20 \times 10=560 \mathrm{~m} 2$
$>$ The heat exchange coefficient K is obtained from calculations heat transfers that would require a full webinar. (Polyurethane foam panels $\mathrm{K}=\mathbf{0 . 2 2} \mathbf{~ W} / \mathbf{m} 2 . K$ ).
$>$ The temperature outside the room is $40^{\circ} \mathrm{C}$
$>$ The temperature to be maintained inside is $2^{\circ} \mathrm{C}$.
This gives a $\Delta \theta$ of $38^{\circ} \mathrm{C}$
$Q$ walls $=0.22 \times 560 \times 38 \times 84400$ $=395,127,040 \mathrm{~J}$
$=395150 \mathrm{~kJ}$

## Thermal balance

## Calculation of the various balance sheet items

## CARACTÉRISTIQUES THERMIQUES

La connaissance de la différence de température entre l'intérieur et l'extérieur de l'enceinte ( $\Delta T$ ) airsi que le choix des déperditions thermiques admissibles (Q) permettent de déterminer l'épaisseur optimum du panneav en utilisant l'abaque des déperditions.
$Q=$ déperditions thermiques - Valeur préconisée par le DTU 45-1 $8 \mathrm{~W} / \mathrm{m}^{2}$ en positif, $6 \mathrm{~W} / \mathrm{m}^{2}$ en négatif.


| Epaisseur en mm | 60 | 70 | 80 | 90 | 100 | 110 | 120 | 130 | 140 | 150 | 160 | 170 | 180 | 190 | 200 | 210 | 220 |
| :--- | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\left.\mathrm{~W} / \mathrm{m}^{2} .{ }^{\circ} \mathrm{C}\right)$ | 0,43 | 0,37 | 0,32 | 0,29 | 0,25 | $\cdot$ | 0,22 | $\cdot$ | 0,19 | - | 0,16 | $\cdot$ | 0,15 | $\cdot$ | 0,13 | $\cdot$ | $\cdot$ |

## Thermal balance

## Calculation of the various balance sheet items

## Item 2: heat input by the products

The term "products" includes the foodstuffs to be cooled as well as their packaging. When it comes to fresh produce or plants (fruits and vegetables), the heat of respiration must also be added. The latter is due to the humidity that emerges from these products when the temperature drops (only in positive temperature).

## Commodities:

## Positive cold:

For foodstuffs undergoing a drop in temperature without reaching $0^{\circ} \mathrm{C}$, the following formula applies:

$$
\mathrm{Q} \text { commodities }=\mathrm{m} \cdot \mathrm{cp} \cdot(\theta \mathrm{i}-\theta \mathrm{f})
$$

With :
Q foodstuffs: heat input by foodstuffs [kJ] m : mass of products entered per day or per cycle $[\mathrm{kg}]$ cp : mass heat capacity of the products before freezing $[\mathrm{kJ} / \mathrm{kg}]$
$\theta$ i: initial product temperature (before cooling) [ $\left.{ }^{\circ} \mathrm{C}\right]$
$\theta$ f: final product temperature $=$ temperature in the chamber $\left[{ }^{\circ} \mathrm{C}\right]$

## Thermal balance

## Calculation of the various balance sheet items

## Freezing:

If the food undergoes freezing during cooling, the formula becomes more complex:

$$
\mathrm{Q} \text { commodities }=\mathrm{m} . \mathrm{cp} 1 .(\theta \mathrm{i}-\theta \mathrm{c})+\mathrm{m} . \mathrm{lf}+\mathrm{m} . \mathrm{cp} 2 .(\theta \mathrm{c}-\theta \mathrm{f})
$$

With :
Q foodstuffs: heat input by foodstuffs [kJ]
m : mass of products entered per day or per cycle $[\mathrm{kg}]$
cp 1 : Specific heat of products before freezing $[\mathrm{kJ} / \mathrm{kg}]$
cp 2 : Specific heat of products after freezing $[\mathrm{kJ} / \mathrm{kg}]$
lf: Latent heat of freezing of products [ $\mathrm{kJ} / \mathrm{kg}$ ]
$\theta$ i: Initial product temperature (before cooling) $\left[{ }^{\circ} \mathrm{C}\right]$
$\theta \mathrm{c}$ : Product freezing temperature $\left(0^{\circ} \mathrm{C}\right)\left[{ }^{\circ} \mathrm{C}\right]$
$\theta \mathrm{f}$ : Final product temperature (temperature in the chamber) $\left[{ }^{\circ} \mathrm{C}\right]$

## Thermal balance

## Calculation of the various balance sheet items

## Breathing:

For fresh produce and vegetables at positive temperature, the heat of respiration is expressed as follows:

$$
Q_{\text {resp }}=m \cdot l_{\text {resp }} \cdot 0^{-3}
$$

With :
Qresp: heat input due to respiration [kJ]
m : mass of products entered per day or per cycle $[\mathrm{kg}]$ lresp: latent heat of respiration [ $\mathrm{kJ} /$ tonne]

## Thermal balance

## Calculation of the various balance sheet items

| PRODUITS |  | PROPRIETES THERMIQUES |  |  |  |  | CONSERVATION DE PRODUITS FRAIS |  |  | CONGÉLATION ET CONSERVATION DES PRODUITS CONGELÉS |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Point de congèl | Chaleur spécifique en $\mathrm{kJ} / \mathrm{kg}$${ }^{*} \mathrm{C}$ |  | Chaleur latente de congélation en | Chaleur de respiration | Tempér.de conserv. | Humidité relative | Durée de conserv. | Temp pour congèl. | Temp pour conserv | Humidité relative | Durée de conserv |
|  |  | ${ }^{\circ} \mathrm{C}$ | Avant congèl. | Après congèl. | kJ/kg | KJ/kg 24h | ${ }^{\circ} \mathrm{C}$ | \% |  | ${ }^{\circ} \mathrm{C}$ | ${ }^{\circ} \mathrm{C}$ | \% |  |
|  | Cerises | -2,2 | 3,64 | 1,89 | 284,92 | 1,47 à 2,1 | O à -1 | 85 à 90 | 1 à 4 sem. | -55 | -18 |  | 1 année |
|  | Dattes sèches | -15,5 | 1,51 | 1,09 | 67,04 |  | O à -2 | 70 | 4 à 8 mois | -18 |  |  | 1 année |
|  | Fraises | -1,1 | 3,85 | 2,01 | 301,68 | 2,51 à 3,77 | 0 | 85 à 90 | 1 à 5 jours | -55 | -18 |  | 1 année |
|  | Citrons | -2,2 | 3,85 | 1,93 | 297,49 | 1,26 à 3,35 | +9 à +10 | 85 à 90 | 6 à 8 sem. |  |  |  |  |
| FRUITS | Mandarines | -2,2 | 3,98 | 2,14 | 293,3 | 1 à 1,5 | +4 à +7 | 85 à 90 | 3 à 6 sem. |  |  |  |  |
|  | Pommes | -2 | 3,6 | 1,89 | 280,73 | 0,42 à 1,68 | -1 à +3 | 85 à 90 | 2 à 5 mois |  |  |  |  |
|  | Melons | -1,4 | 4,06 | 2,05 | 322,63 | 2,1 à 4,19 | + 4 à +10 | 85 à 90 | 1 à 4 em . |  |  |  |  |
|  | Poires | -1,9 | 3,56 | 1,89 | 280,73 | 0,75 à 0,92 | Oà -1 | 85 à 90 | 1 à 6 mois |  |  |  |  |
|  | Pêches | -1,5 | 3,77 | 1,89 | 293,3 | 0,92 à 1,47 | -là +1 | 85 à 90 | 1 à 4 sem. | -55 | -18 |  | 1 année |
| Pamplemousses |  | -2 | 3,77 | 1,89 | 293,3 | 0,84 à 1,47 | O à +10 | 85 à 90 | 4 à 6 sem. |  |  |  |  |
| Prunes |  | -2,2 | 3,69 | 1,89 | 289,11 | 0,42 à 0,75 | 0 | 85 à 90 | 3 à 4 sem. | -55 | -18 |  | 1 année |

## Thermal balance

## Calculation of the various balance sheet items

Packaging:

$$
\text { Qemb = m.cp. }(\theta \mathbf{i}-\theta \mathrm{f})
$$

With :
Qemb: heat input by packaging [kJ] m : mass of packaging [kg] cp : specific thermal capacity of packaging $[\mathrm{kJ} / \mathrm{kg}]$ $\theta \mathrm{i}$ : initial packaging temperature $\left[{ }^{\circ} \mathrm{C}\right]$ $\theta \mathrm{f}$ : final packaging temperature $\left[{ }^{\circ} \mathrm{C}\right]$

## Thermal balance

## Calculation of the various balance sheet items

## Total products :

The product item can be summarized as follows:
Qproducts = Qcommodities + Qresp + Qemb

Example :
Data: Mass of apples to introduce: m apples $=30$ tonnes
Chamber capacity: 300 tons
Specific heat of apples: cp apples $=3.6 \mathrm{~kJ} / \mathrm{kg} . \mathrm{K}$
Heat of respiration of apples at $2^{\circ} \mathrm{C}$ : C resp $=1.3 \mathrm{~kJ} / \mathrm{kg}$ at $20^{\circ} \mathrm{C}$ : Cresp $=53 \mathrm{KJ} /$ tonne
Mass of wooden crates: m crates $=20 \mathrm{~kg}$
Specific heat of wood: cp wood $=2.75 \mathrm{~kJ} / \mathrm{kg} . \mathrm{K}$
It is considered that the products return to $20^{\circ} \mathrm{C}$ to be cooled to $2^{\circ} \mathrm{C}\left(\Delta \theta=18^{\circ} \mathrm{C}\right)$
Q products $=\mathrm{Q}$ foodstuffs +Q resp +Q packaging
$Q$ products $=(30000 \times 3.6 \times 18)+[(30000 \times 1.3)+(270 \times 53)]+(20 \times 2.75 \times 18)$
Q products $=1,998,300 \mathrm{~kJ}$

## Thermal balance

## Calculation of the various balance sheet items

## Item 3: heat input by ventilation

The evaporators installed in the enclosure to be refrigerated are generally equipped with fans. The thermal losses of the electric motors constitute a heat input to be taken into account in the balance:

$$
\mathbf{Q} \text { ventil }=\mathbf{P} \text { abs } . \mathbf{t}
$$

With :
Q ventil: heat input by ventilation [kJ]
Pabs: power absorbed by fan motors [kW]
t : fan operating time [s]
The running time of the fans varies depending on the application:
General case : the running time of the fans is identical to the running time of the compressors.
Case of a room with a significant height: the fans remain permanently powered to avoid stratification of the air temperature. The operating time of the fans is then 24 hours (86400 seconds).
Case of a tunnel: the ventilation is on throughout the cycle: the operating time to be taken into account is then the cycle time.

## Thermal balance

## Calculation of the various balance sheet items

## Fan power input is calculated as follows:

With :
Qv: fan volume flow [m3/s]
$\Delta \mathrm{P}$ : pressure supplied by the fan [Pa]

$$
\mathrm{P}_{\mathrm{abs}}=\frac{\mathrm{Q}_{\mathrm{V}} \cdot \Delta \mathrm{P}}{\eta}
$$

$\longrightarrow 200$ to 400 Pa for a cold room up to 600 Pa for a tunnel
$\eta$ : overall efficiency of the fan (between 0.5 and 0.6 )
The flow rate Qv of the fan is obtained from another formula:
With :
$\tau \mathrm{B}:$ mixing rate in volumes per hour
15 to $30 \mathrm{vol} / \mathrm{h}$ for cold storage rooms at negative temperature
$\longrightarrow 20$ to $40 \mathrm{vol} / \mathrm{h}$ for cold storage rooms at positive temperature
$\longrightarrow 300$ to $900 \mathrm{vol} / \mathrm{h}$ for tunnels
$Q_{V}=\frac{\tau_{B} \cdot V_{C F}}{3600}$
$\longrightarrow$ VCF: volume of cold room or tunnel [m3]
For tunnels, this flow can also be obtained by setting an air speed, while knowing the air passage section:
$\mathrm{Qv}=\mathrm{v} . \mathrm{S}$
with: $\quad \mathrm{v}$ : air velocity $[\mathrm{m} / \mathrm{s}]$
S: air passage section [m2]

## Thermal balance

## Calculation of the various balance sheet items

## Example :

- The cold storage room is at positive temperature
- $30 \mathrm{vol} / \mathrm{h}$ brewing rate
- Chamber volume of 1200 m 3

The flow is:

$$
Q_{v}=\frac{30 \times 1200}{3600}=10 \mathrm{~m}^{3} / \mathrm{s}
$$

- Selected fan pressure differential of 300 Pa
- Fan efficiency is 0.6.

The power of the fan will then be:

$$
P_{\text {abs }}=\frac{10 \times 300}{0,6}=5000 \mathrm{~W}
$$

On a 24 -hour day, the compressors are considered to operate for 18 hours (depending on the regulation).
The operating time of the fans is then identical, which makes it possible to calculate the load due to ventilation:
$Q$ ventil $=5 \times 18 \times 3600=324000 \mathrm{~kJ}$

## Thermal balance

## Calculation of the various balance sheet items

## Item 4: heat input due to air renewal

The renewal of air corresponds to the openings of the doors of the cold room, which constitute an additional heat contribution.

$$
\text { QRA = m air } \cdot(\text { he }-\mathrm{hi})
$$

With :
QRA: heat gain due to air renewal [kJ]
m air: mass of fresh air entering the chamber $[\mathrm{kg}]$
he: outside air enthalpy $[\mathrm{kJ} / \mathrm{kg}]$
hi: indoor air enthalpy [kJ/kg]

## Thermal balance

## Calculation of the various balance sheet items

It is necessary to know how to estimate the mass of incoming air. For this, we use the notion of air renewal rate ( $\mathrm{vol} / \mathrm{h}$ ), or number of air renewals per $24 \mathrm{~h}(\mathrm{vol} / \mathrm{d})$.

With :

$$
m_{\text {air }}=\frac{V_{\text {air }}}{v_{i}^{\prime \prime}}=\frac{\mathrm{nr} \cdot \mathrm{~V}_{\mathrm{CF}}}{\mathrm{v}^{\prime \prime}{ }_{i}}
$$

Vair: volume of incoming air [m3]
v "i: specific volume of indoor air [ $\mathrm{m} 3 / \mathrm{kg}$ ]
VCF: cold room volume [m3]
nr : number of air changes [ $\mathrm{vol} / \mathrm{d}$ ]
This number of air changes corresponds to the number of times the
volume of the chamber is renewed in one day, i.e. 24 hours.
A curve makes it possible to estimate this number according to the volume of the chamber:

## Thermal balance

## Calculation of the various balance sheet items



## Thermal balance <br> Calculation of the various balance sheet items

## Example :

The apple preservation chamber is to be considered as a long preservation chamber (multiply nr by $0.6)$. Knowing its volume of 1200 m 3 , we get $\mathrm{nr}=2.2 \times 0.6=1.32$.
The humidity maintained in the room is $90 \%$.
The specific volume of air at $2^{\circ} \mathrm{C}$ and $90 \%$ is $0.783 \mathrm{~m} 3 / \mathrm{kg}(\mathrm{as})$, which gives an air mass of:

$$
\mathrm{m}_{\text {air }}=\frac{1,32 \times 1200}{0,787}=2012,7 \mathrm{~kg}
$$

he $=100.69 \mathrm{~kJ} / \mathrm{kg}$ (as)
$\mathrm{hi}=11.84 \mathrm{~kJ} / \mathrm{kg}(\mathrm{as})$
QRA $=2012,7 \times(100,69-11,84)=178828 \mathrm{~kJ}$

## Thermal balance Calculation of the various balance sheet items

Example :
The humidity maintained in the room is $90 \%$. The specific volume of air at $2^{\circ} \mathrm{C}$ and $90 \%$ is $0.783 \mathrm{~m} 3 /$ kg (as),
Consider that the outside air is at $40^{\circ} \mathrm{C} / 50 \%$.
he $=100.69 \mathrm{~kJ} /$
$\mathrm{kg}(\mathrm{as})$
$\mathrm{hi}=11.84 \mathrm{~kJ} / \mathrm{kg}(\mathrm{as})$


## Thermal balance

## Calculation of the various balance sheet items

## Item 5: heat input by machines Qmach $=\mathbf{N} . \mathbf{P} . t$

With :
Q mach: heat input by machines [kJ]
N : number of machines
P : thermal power released by the devices $[\mathrm{kW}]$
t : operating time [ s ]
Example:
Forklift power $=4.4 \mathrm{KW}$
Stay in the room 2 hours
$Q$ mach $=4,4 \times(2 \times 3600)=31680 \mathrm{KJ}$

## Thermal balance

## Calculation of the various balance sheet items

## Item 6: heat input by lighting

$$
Q \text { ecl = P écl . t }
$$

With :
Q ecl: heat input by the lighting [kJ]
P ecl: thermal power released by the lighting $[\mathrm{kW}]$
t : calculation time [s]
The power of the lighting is around 5 to $20 \mathrm{~W} / \mathrm{m} 2$. This value must be multiplied by the floor area and divided by 1000 to obtain the power in kW .

## Example:

Room area $=200 \mathrm{~m} 2$
Lighting power is estimated at $10 \mathrm{~W} / \mathrm{m} 2$
Stay in the room 2 hours

$$
\begin{aligned}
\mathrm{Q} \text { ecl } & =200 \times 10 \times(2 \times 3600)=14400000 \mathrm{~J} \\
& =14400 \mathrm{KJ}
\end{aligned}
$$

## Thermal balance

## Calculation of the various balance sheet items

## Item 7: heat input by the occupants Qpers $=\mathbf{N}$. P/pers . t

With :
Qpers: heat input by the occupants [kJ]
N : number of people
$\mathrm{P} /$ pers $=$ power per person $[\mathrm{kW}]$
t : occupation time [s]
Example:
Number of people $=2$ people
Power per person $=0.258 \mathrm{KW}$
Stay in the room 2 hours
$Q$ ecl $=2 \times 0,258 \times(2 \times 3600)=3715$ に。

| Température de <br> la chambre $\left({ }^{\circ} \mathrm{C}\right)$ | Puissance dégagée <br> par personne $(\mathbf{W})$ |
| :---: | :---: |
| 10 | 210 |
| 5 | 240 |
| 0 | 270 |
| -5 | 300 |
| -10 | 330 |
| -15 | 360 |
| -20 | 390 |
| -25 | 420 |

## Thermal balance

## Sum of contributions from all positions

Once all the items have been calculated, the quantities of heat must be added together to obtain the total balance, in kJ . Items 5, 6 and 7 are often negligible in relation to the other items, and difficult to calculate. In general, we will take a percentage of the previous contributions to estimate the quantity of heat released by these three stations from 5 to $10 \%$ of total Q
$\mathrm{Q}_{\text {total }}=\sum \mathrm{Q}$
Qtotal = Qwalls + Qproducts + Qventil + QRA + Qmach + Qecl

+ Qpers


## Thermal balance

## Sum of contributions from all items

## Example :

Qwalls + Qproducts + Qventil + QRA + Qmach + Qproducts + Qpers
$=1,998,300+324,000+178,828+31,680+14,400+3,715$ the overall sum of the contributions:

$$
Q_{\text {total }}=2550923 \mathrm{~kJ}
$$

## Thermal balance

## Calculation of cooling capacity

The final phase of the calculation of the balance consists in dividing the total quantity of energy by the operating time of the compressors to obtain the cooling capacity to be installed.

$$
\Phi_{0}=\frac{\mathrm{Q}_{\text {total }}}{\mathrm{t}_{\mathrm{CP}}}
$$

Attention ! tcp is the running time of the compressors which is used to calculate the power of the equipment. Not to be confused with the base time of the calculation which is used to assess the heat input over the entire refrigeration or storage period.

## Thermal balance

## Calculation of cooling capacity

Indeed, the compressors do not work for the entire duration of use.
If it is a cold room, they will be considered to operate between 16 and 20 hours a day (depending on how often the doors are opened).

On the other hand, for a tunnel, the compressors operate for the entire duration of the cycle. In this case, tcp = base time of the calculation

## Example :

In the case of a cold storage room, it will be considered that the compressors operate on average 18 hours a day.
The balance is then obtained as follows: $\mathbf{2 , 5 5 0 , 9 2 3} / \mathbf{1 8} \times 3600=\mathbf{3 9 . 3 6 6} \mathrm{KJ} / \mathbf{s}$ Always round up:

$$
\Phi 0=40 \mathrm{~kW}
$$

$$
\begin{gathered}
\text { شion } \\
\text { Merci } \\
\text { Thanks } \\
\text { Gracias } \\
\text { Obrigado }
\end{gathered}
$$

## 2 QUESTIONS / REPONSES

